

TRANSVERSE VIBRATIONS OF RECTILINEAR PLATES

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The transverse vibrations of a rectangular plate have been considered very fully by Kohn and his co-workers.¹ Elliptical plates have been considered by Mindlin² and Timoshenko.³ The complete analytical solution of free vibrations of a homogeneous isotropic rectangular plate, given by Mary D. Wells,⁴ is the only published complete analytical study of the square plate. The present paper is a study of the square plate by the method of separation of variables, as first suggested by Rayleigh⁵ and Reissner,⁶ and is similar to the study of the rectangular modes of a square or rectangular plate given by Kohn et al.,¹ $\sqrt{\frac{11}{12}}$ which is the length of a side, ρ_p the average density, D the flexural rigidity, $\nu = \frac{1}{2}(1 + \nu_1 + \nu_2)$, ν_1 being Young's modulus in the x direction, and ν_2 the Poisson's ratio.

If the plate vibrates in a normal mode w of the form $W(x, y, t) = w$ we know that W is a function of x and y satisfying the equation⁷

$$\nabla^4 W + \left(\frac{1}{\rho_p} - \frac{1}{\rho_p}\right) W = \frac{1}{D} \ddot{W} = k^2 W \quad (1)$$

The eigenvalues k_p are to be determined by adapting the solution to the specific boundary conditions.

For the case of all edges clamped, and w along a fixed axis, and by suitable choice of origin at the point under consideration, we know that the boundary conditions on all four edges are⁸

$$\frac{\partial^2 w}{\partial x^2} = \frac{\partial^2 w}{\partial y^2} = \frac{\partial w}{\partial x} = \frac{\partial w}{\partial y} = 0 \quad (2)$$

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ν being the modulus of elasticity. Silverman⁹ has shown that these conditions are satisfied by a particular system of circular orthogonal coordinates.

At a clamped edge the boundary conditions are

At a supported edge these conditions become

$$W = 0, \sigma \nabla^2 W = \sigma \left(\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) W = (\sigma - 1) \frac{\partial^2 W}{\partial n^2}. \quad (2.4)$$

For rectilinear plates the distinction between s and μ disappears.

SUPPORTED EDGES

To begin with we take the case of a rectilinear plate whose edges are all supported. Rayleigh² has given the solution for a rectangular plate. It can be shewn that the problem of the rectilinear plate can be reduced to that of a rectilinear membrane.

For the transverse vibrations of a membrane the differential equation to be satisfied by W is

$$\nabla^2 W + k^2 W = 0 \quad (3)$$

with the condition that $W = 0$ over the boundary. Operating (3) by ∇^2 we get

$$\nabla^4 W = k^4 W \quad (4)$$

which is the same as (1). Any solution of (3), therefore, also satisfies (4) over the boundary $W = 0$, and hence from (3) $\nabla^2 W = 0$. We can now shew that (2.4) also reduces to this condition.

As $W = 0$ over the boundary, we have, differentiating along the boundary $\partial W / \partial s = 0$, $\partial^2 W / \partial s^2 = 0$. Also

$$\nabla^2 W = \frac{\partial^2 W}{\partial s^2} + \frac{1}{\rho} \frac{\partial W}{\partial n} + \frac{\partial^2 W}{\partial n^2},$$

which for a rectilinear boundary ($\rho = \infty$) reduces to

$$\nabla^2 W = \frac{\partial^2 W}{\partial s^2} + \frac{\partial^2 W}{\partial n^2} = \frac{\partial^2 W}{\partial n^2}$$

Thus (2.4) also becomes $\nabla^2 W = 0$. For a simply supported rectilinear boundary the boundary conditions can therefore be put in the form $W = \nabla^2 W = 0$.^{9, 10}

Hence, if we know the solution for a membrane, the corresponding solution for a plate can be easily written down. In fact the form of W given by Rayleigh for a rectangular plate is the same as that for a membrane.

TRIANGULAR PLATES

We have already discussed in another paper¹¹ the transverse vibrations of triangular membranes. For triangular plates it is therefore quite sufficient to give the main results,

Rectangular plate—The corresponding W is given by

$$W = \frac{(2m+1)\pi x}{2a} \cos \frac{(2n+1)\pi y}{2b} \cos \frac{(2m+1)\pi x}{2a} \cos \frac{(2n+1)\pi y}{2b} \quad (5)$$

giving the frequency $\omega = \frac{m^2+n^2}{a^2} \sqrt{D/m_0}$. The frequency is constant.

$$\begin{aligned} P_0 &= \frac{1}{4} \frac{\pi^2}{a^2} [(2m+1)^2 + (2n+1)^2] \sqrt{D/m_0} \\ P_0 &= \frac{1}{2a^2} [(2m+1)^2 + (2n+1)^2] \sqrt{D/m_0} \end{aligned} \quad (6)$$

The general case is given by

$$\begin{aligned} P_0 &= \frac{1}{2a^2} \sqrt{D/m_0} \\ W_{xy} &= \cos \frac{m\pi x}{a} \cos \frac{n\pi y}{b} \cos \frac{m\pi x}{a} \cos \frac{n\pi y}{b} \end{aligned} \quad (7)$$

The relations for the non-symmetrical situation. The symmetrical are given by

$$W = \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} \quad (8)$$

Equilateral plate—The corresponding results are given by

$$\begin{aligned} W &= 2 \sin \frac{(m+n)\pi x}{a} \cos \frac{(m-n)\pi y \sqrt{3}}{a} \\ &= 2 \sin \frac{(2m+n)\pi x}{a} \cos \frac{m\pi y \sqrt{3}}{a} + 2 \sin \frac{(2n+m)\pi x}{a} \cos \frac{m\pi y \sqrt{3}}{a} \end{aligned} \quad (9.1)$$

$$P_0 = \frac{1}{2} \frac{\pi^2}{a^2} [m^2 + mn + n^2] \sqrt{D/m_0} \quad (9.2)$$

$$P_0 = \frac{1}{2} \frac{\pi^2}{a^2} \sqrt{D/m_0} \quad (9.3)$$

$$W_x = 2 \sin \frac{(2m+n)\pi x}{a} \cos \frac{m\pi y \sqrt{3}}{a} \quad (9.4)$$

the sides of the plates being $\sqrt{3}a$, $\cos = \sqrt{3} \sqrt{3}$.

Triangles triangular plate containing an angle of 120°—In this case the results are

$$\begin{aligned} W &= 2 \sin \frac{(m+n)\pi x}{a} \sin \frac{(m+n+1)\pi y \sqrt{3}}{a} \\ &= 2 \cos \left(2m+1 - \frac{2n+1}{2} \right) \frac{\pi x}{a} \cos \frac{(2n+1)\pi y \sqrt{3}}{2a} \\ &+ 2 \cos \left(2n+1 - \frac{2m+1}{2} \right) \frac{\pi x}{a} \cos \frac{(2m+1)\pi y \sqrt{3}}{2a} \end{aligned} \quad (10.1)$$

$$\frac{p}{\pi} = \frac{\pi}{a^2} [(2m+1)^2 + (2m+1)(2n+1) + (2n+1)^2] \sqrt{\frac{D}{m_0}} \quad (10.2)$$

$$\frac{p_0}{\pi} = \frac{7\pi}{2a^2} \sqrt{\frac{D}{m_0}} \quad (10.3)$$

$$W_0 = 2 \sin \frac{2\pi x}{a} \sin \frac{\pi y \sqrt{3}}{a} - 2 \cos \frac{5\pi x}{2a} \cos \frac{\pi y \sqrt{3}}{2a} \\ + 2 \cos \frac{\pi x}{2a} \cos \frac{3\pi y \sqrt{3}}{2a} \quad (10.4)$$

In this case the modes are all symmetrical. The sides are given by $x = a$, $y = x\sqrt{3}$, $y + x\sqrt{3} = 2a/\sqrt{3}$.

Right-angled triangle containing an angle of 60°.—If the sides are taken as $x = a$, $y = a/\sqrt{3}$, $y = x\sqrt{3}$, the results given in (10) hold good in this case as well.

RHOMBUS AND A REGULAR HEXAGON

If we take the sides of the rhombus, which contains an angle of 120°, as $x = 0$, $x = a$, $y = x/\sqrt{3}$, $y = x/\sqrt{3} + 2a/\sqrt{3}$, and those of the regular hexagon as $x = \pm a$, $y = \pm x/\sqrt{3} \pm 2a/\sqrt{3}$, the solution in (9) holds good.¹²

In all the above cases we find that the frequency is proportional to $\frac{1}{a^2} \sqrt{\frac{D}{m_0}}$.

CLAMPED EDGES

Square-plate.—If we take the sides of the plate as $y = \pm x \pm 2a$, we find that the solutions given in (6) and (7) also satisfy the condition $\partial W/\partial n = 0$ over the boundary. Thus the conditions $W = 0$, $\partial W/\partial n = 0$ for a clamped edge are satisfied on all the sides of the plate. It is found that these conditions are also satisfied on the lines $y = \pm x$. Hence (6) and (7) give the symmetrical vibrations.

If we use this solution for a right-angled isosceles plate we find that the conditions for a clamped edge are satisfied on the equal sides $y = \pm x$; but that on the edge $x = a$ the conditions for a supported edge are only satisfied. Hence it may be used when the equal sides of the plate are clamped and the base is supported.

Free vibrations of a square-plate.—If the edges are free the boundary condition (2.1) must also be satisfied. Since $\nabla^2 W = -k^2 W$ and

$$\begin{aligned} \frac{\partial^3 W}{\partial n \partial s^2} &= \frac{1}{2} \frac{\partial}{\partial n} \left(\frac{\partial}{\partial x} \pm \frac{\partial}{\partial y} \right)^2 W = \frac{1}{2} \frac{\partial}{\partial n} \left[\nabla^2 W \pm 2 \frac{\partial^2 W}{\partial x \partial y} \right] \\ &= \frac{1}{2} \frac{\partial}{\partial n} \left[-k^2 W \pm \frac{\pi^2}{2a^2} (2m+1)(2n+1) W \right], \\ &= \frac{1}{2} \left[-k^2 \pm \frac{\pi^2}{2a^2} (2m+1)(2n+1) \right] \frac{\partial W}{\partial n}. \end{aligned}$$

We see that (2.1) is also satisfied on all the sides of the plate. The solution in (6) and (7) can therefore also be used for the free symmetrical vibrations of a square plate.

As in the case of clamped edges, this solution can be used for a right-angled isosceles triangle whose equal sides are free but whose base is supported.

SUMMARY

The problem of the vibrations of a rectilinear plate with supported edges can be reduced to the corresponding problem of a vibrating membrane. Exact solutions are given for a number of triangular plates. The free and clamped vibrations of a square and a right-angled isosceles triangular plate have also been discussed.

REFERENCES

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| 1. Kirchhoff | .. <i>Journal für Math.</i> (Crelle), 1850, 40. |
| 2. Rayleigh | .. <i>Theory of Sound</i> , 1894, Vol. I, Chap. X. |
| 3. Mathieu | .. <i>Journal de Math.</i> (Liouville), 1869, 14, (Sér. 2). |
| 4. Barthélémy | .. <i>Toulouse Mém. de l' Acad.</i> , 1877, 9. |
| 5. Mary D. Waller | .. <i>Proc. Phy. Soc.</i> , 1939, 51, 831 ; 1941, 53, 35. |
| 6. Rayleigh | .. <i>Phil. Mag.</i> , 1911, 22, 225. |
| 7. Ritz | .. <i>Ann. Phys.</i> , Leipzig, 1909, 28, 737. |
| 8. Timoshenko | .. <i>Vibration Problems in Engineering</i> , 1937, 2nd ed., § 70. |
| 9. Stevenson | .. <i>Phil. Mag.</i> , 1943, 34, 110. |
| 10. Timoshenko | .. <i>Theory of Plates and Shells</i> , 1940, 100. |
| 11. Seth | .. <i>Proc. Ind. Acad. Sci. (A)</i> , 1940, 12, 487. |
| 12. ————— | .. <i>Ibid.</i> , 1941, 13, 390. |